

# Exam II Study Guide

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## 1 Spring-Mass Systems

### 1.1 Modeling

$$mu'' + \gamma u' + ku = F(t)$$

where  $u$  is the displacement from the equilibrium position,  $m$  the mass,  $\gamma$  the damping constant and  $F(t)$  the external force.  $k$  is the spring constant which can be determined from the equilibrium condition  $mg = kL$ , where  $mg$  is the weight and  $L$  is the elongation of the spring at equilibrium position.

### 1.2 Undamped Free Vibrations

$$mu'' + ku = 0$$

General solution:  $u = A \cos \omega_0 t + B \sin \omega_0 t = R \cos(\omega_0 t - \delta)$

Amplitude:  $R = \sqrt{A^2 + B^2}$

Natural Frequency:  $\omega_0 = \sqrt{k/m}$

Period:  $T = 2\pi/\omega_0$

Phase:  $\delta$ , such that  $R \cos \delta = A$  and  $R \sin \delta = B$

### 1.3 Damped Free Vibrations

$$mu'' + \gamma u' + ku = 0$$

The system is losing its energy so every solution asymptotically approaches zero as  $t \rightarrow \infty$ , regardless of the type of damping.

#### 1.3.1 Over-damping and Critical damping

The characteristic equation may have two (negative) real roots, repeated (negative) real roots or two complex roots (with negative real parts).

**Over-damping:**  $\gamma^2 - 4km > 0$ , two real roots, solution does not oscillate

**Critical damping:**  $\gamma^2 - 4km = 0$ , repeated roots, solution does not oscillate

In both cases, the solution does not oscillate and can cross the equilibrium position at most once.

### 1.3.2 Under-damping

$\gamma^2 - 4km < 0$ , two complex roots, solution oscillates with decreasing amplitude and crosses the equilibrium position infinitely many times.  
General solution:  $u = e^{-\gamma t/2m}(A \cos \mu t + B \sin \mu t) = Re^{-\gamma t/2m} \cos(\mu t - \delta)$ ,  
Quasi frequency:  $\mu = \sqrt{4km - \gamma^2}/2m$   
Quasi period:  $T_d = 2\pi/\mu$

### 1.4 Undamped Forced Vibration

$$mu'' + ku = F_0 \cos \omega t$$

Resonance:  $\omega_0 = \omega$ , where  $\omega_0 = \sqrt{k/m}$  is the natural frequency

## 2 Laplace Transform

### 2.1 Definition

$$F(s) = \mathcal{L}\{f(t)\} = \int_0^{\infty} e^{-st} f(t) dt$$

Additivity:  $\mathcal{L}\{af(t) \pm bg(t)\} = a\mathcal{L}\{f(t)\} \pm b\mathcal{L}\{g(t)\}$ .

### 2.2 Solving Initial Value Problems

$$y'' + py' + qy = g(t), y(0) = y_0, y'(0) = y'_0$$

Step 1: Laplace transform:

$$s^2 Y(s) - sy(0) - y'(0) + p(sY(s) - y(0)) + qY(s) = \mathcal{L}\{g(t)\}$$

where  $Y(s) = \mathcal{L}\{y\}$

Step 2: Plug in initial conditions and solve for  $Y(s)$  from the equation above.

Step 3: Take inverse transform and find  $y = \mathcal{L}^{-1}\{Y(s)\}$

### 2.3 Step Functions

#### 2.3.1 Unit Step Functions

$$u_c(t) = u(t - c) = \begin{cases} 0, & t < c \\ 1, & t \geq c \end{cases}$$

#### 2.3.2 Piecewise Defined Functions as Step Functions

$$f(t) = \begin{cases} f_1(t), & t < t_1 \\ f_2(t), & t_1 \leq t < t_2 \\ \vdots \\ f_n(t), & t_{n-1} \leq t \end{cases} = f_1(t) + u(t-t_1)(f_2(t) - f_1(t)) + \cdots + u(t-t_{n-1})(f_n(t) - f_{n-1}(t))$$

Also need to know: Evaluation of a step function.

### 2.3.3 Laplace Transform of Step Functions

$$\begin{aligned}\mathcal{L}\{u(t-c)\} &= e^{-cs}/s \\ \mathcal{L}\{u(t-c)f(t)\} &= e^{-cs}\mathcal{L}\{f(t+c)\}\end{aligned}$$

### 2.3.4 Inverse Transform

To find the inverse transform for  $e^{-cs}F(s)$ , first find  $f(t) = \mathcal{L}^{-1}\{F(s)\}$ . Then

$$\mathcal{L}^{-1}\{e^{-cs}F(s)\} = u(t-c)f(t-c)$$

## 2.4 Impulse Functions

### 2.4.1 Definition

Unit impulse function  $\delta(t)$  is an idealized function with the following properties:

$$\delta(t) = 0 \text{ if } t \neq 0; \quad \int_{-\infty}^{\infty} \delta(t)dt = 1$$

More properties:

$$\int_{-\infty}^{\infty} f(t)\delta(t-t_0) = f(t_0) \quad \text{and} \quad f(t)\delta(t-t_0) = f(t_0)\delta(t-t_0)$$

### 2.4.2 Laplace Transform

$$\begin{aligned}\mathcal{L}\{\delta(t-t_0)\} &= e^{-st_0} \\ \mathcal{L}\{\delta(t-t_0)f(t)\} &= f(t_0)e^{-st_0}\end{aligned}$$

## 3 Homogeneous Linear Systems with Constant Coefficients

### 3.1 Solving the Equation $\mathbf{x}' = \mathbf{A}\mathbf{x}$

Step 1: Look for eigenvalues: set  $\det(\mathbf{A} - \lambda\mathbf{I}) = 0$  and solve for  $\lambda$ .

Step 2: Look for eigenvectors: Plug in the  $\lambda$  you get in step 1, find a nonzero vector  $\xi$  such that  $(\mathbf{A} - \lambda\mathbf{I})\xi = 0$

Step 3: 3 cases here depending on the roots of the equation  $\det(\mathbf{A} - \lambda\mathbf{I}) = 0$

#### 3.1.1 Different Real Roots

General solution:  $\mathbf{x} = c_1e^{\lambda_1 t}\xi_1 + c_2e^{\lambda_2 t}\xi_2$

Phase portrait: Two straight lines; Critical point (origin) is a node or a saddle point.

### 3.1.2 Complex Roots

Take one complex root  $a + bi$  and its eigenvector  $\xi$ . Find real valued vectors  $\xi_1$  and  $\xi_2$  such that

$$e^{(a+bi)t}\xi = e^{at}(\cos bt + i \sin bt)\xi = \xi_1 + i\xi_2$$

Then the general solution is  $\mathbf{x} = c_1\xi_1 + c_2\xi_2$

Phase portrait: Spiral or center (if the complex roots have zero real parts)

### 3.1.3 Repeated Roots

Find a vector  $\eta$  such that  $(\mathbf{A} - \lambda\mathbf{I})\eta = \xi$

The general solution is  $\mathbf{x} = c_1e^{\lambda t}\xi + c_2e^{\lambda t}(t\xi + \eta)$

Phase portrait: Improper node (one eigenvector) or proper node (every vector is an eigenvector)

## 3.2 Phase Portrait and Stability

A detailed summary with pictures is provided in Section 9.1. You may find Table 9.1.1 on page 494 and Figure 9.1.9 on page 497 helpful.

## 3.3 Transform between Higher Order Equations and Systems of First Order Equations

An arbitrary  $n$ th order equation  $y^{(n)} = F(t, y, y', \dots, y^{(n-1)})$  can be transformed into a system of  $n$  first order equations. Set  $x_1 = y, x_2 = y', \dots, x_n = y^{(n-1)}$ . The system is

$$\begin{cases} x'_1 = x_2 \\ x'_2 = x_3 \\ \vdots \\ x'_{n-1} = x_n \\ x'_n = F(t, x_1, x_2, \dots, x_n) \end{cases}$$

## 4 Nonlinear Systems $\begin{cases} x' = F(x, y) \\ y' = G(x, y) \end{cases}$

Critical points: can be solved from  $\begin{cases} F(x, y) = 0 \\ G(x, y) = 0 \end{cases}$ .

Type of the critical point  $(x_0, y_0)$ : determined by the linear system with the matrix  $\begin{pmatrix} \partial F/\partial x(x_0, y_0) & \partial F/\partial y(x_0, y_0) \\ \partial G/\partial x(x_0, y_0) & \partial G/\partial y(x_0, y_0) \end{pmatrix}$ . For correspondence between linear and nonlinear systems, see Table 9.3.1 on Page 513.

**This study guide may contain mistakes or typos. If you find any, please tell me as soon as possible.**